Electromagnetic multiple-disc clutch Type 521



Drive elements are our world.



Electromagnetic multiple-disc clutch - Type 521



Characteristics and features

- suitable for torque transmission with increasing differential speed between the drive elements
- high torque transfer despite compact dimensions
- designs up to 2100 Nm possible
- high switching frequency due to optimized heat dissipation
- low maintenance due to slip-ring-free power supply and lifetime-lubricated bearings
- negligible wear due to special friction lining
- oil running or dry running
- suitable for applications in harsh environments
- reduced shift speeds due to adapted control













Mönninghoff power transmission represents an infinite variant diversity that is applied by all areas of modern mechanical engineering.

Our technologies are mostly designed to operate under extreme conditions. We offer high precision products for medical robotics, fail-proof security for aerospace technology or synchronization soultions for the packaging or printing industry.

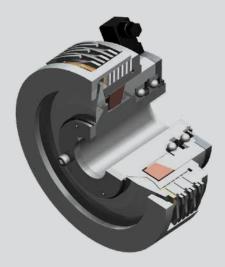
We thus address customers who have the highest standards for their own machines or systems. To them, we can offer highly complex, application-specific solutions.

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Match code

Mönninghoff multiple-disc clutches are indicated by the following match code:



521.A.B.C

- A clutch size
- **B** operating mode
- C length of drive ring

Other individual characteristics:

- voltage
- bore size with keyway

According to these characteristics, we design individual solutions concerning transmitted torque, engaging behavior or rotation speed.

Our engineers can assist with finding an application-specific clutch at any time. Together, we can develop individual and innovative solutions for extreme operating conditions.

Ordering example

Mönninghoff electromagnetic multiple-disc clutch Type 521.21.1.1

Operating mode dry running Voltage 24 Vdc

Bore size d 30 mm H7, keyway acc to. DIN 6885/1

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Clutch size

The selection of the correct size of a Mönninghoff electromagnetic multiple-disc clutch is determined by the requiered torque as well as the shift work.

According to the required torque

$$M_S \ge M_{erf}$$

According to the shift work

$$E_h \leq Q_h$$

The clutch must transfer load and acceleration torque $(M_L; M_b)$. The required safety is obtained by using a corresponding safety factor (K).

$$M_{erf} = (M_b \pm M_L) \cdot K$$

$$M_b = \frac{I \cdot \Delta n}{9,55 \cdot t} [Nm]$$

$$Q_h = Q \cdot k_1 \cdot k_2 [Nm]$$

$$E_h = \frac{I \cdot (\Delta n)^2 \cdot Z}{182.4} [Nm]$$

If the load and acceleration torque cannot be determined, the required torque can be derived from the driving power, taking the required safety into cosideration.

$$M_{erf} = 9550 \cdot \frac{P}{n} \cdot K \text{ [Nm]}$$

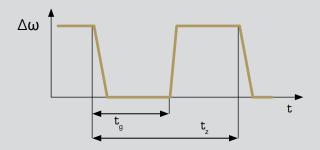
| M_{erf} | = required torque | Р | = driving power [kW] |
|----------------------------|--|---------|---------------------------------------|
| $M_{\scriptscriptstyle b}$ | = acceleration torque | K | = safety factor [1,2 to 4] |
| M_{s} | = shift torque | 1 | = moment of inertia [kgm²] |
| M_{L} | = output load torque | Z | = number of shift operations per hour |
| n | = speed of rotations [min ⁻¹] | Q | = amount of heat |
| Δn | = differential speed of rotations [min ⁻¹] | E_{h} | = shift energy per hour [Nm] |
| k ₁ | = correction factor | t | = acceleration time [sec] |
| k_2 | = correction factor | | based on t ₁ |

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Determination of shift work

The energy that is lost in the clutch depends on the shift curve and the shift frequency. The correction factors for the permissible shift work per hour Q_h can be derived from the tables and graphs.

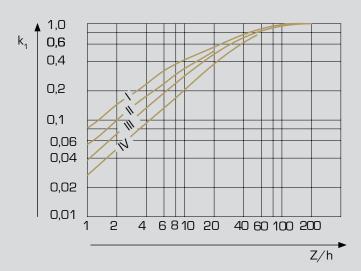


Course of a shift cycle

t_a = time during which the clutch is closed

t, = total cycle time

 $\Delta \omega$ = differential angular velocity



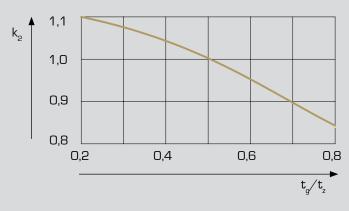
Correction factor $\boldsymbol{k}_{\!\scriptscriptstyle 1}$ as a function of the shift frequency per hour

I valid for 521.32 - 521.33

II valid for 521.24 - 521.28

III valid for 521.21 - 521.22

IV valid for 521.16



Correction factor k, as function t /t,



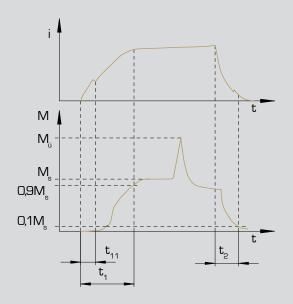
| Size | | | 16 | 21 | 22 | 24 | 26 | 28 | 31 | 32 | 33 |
|------------------|--|------------|------|------|------|------|------|------|------|------|------|
| Amount of heat Q | dry running: bad ventilation oil lubrication: oil spray | 10⁵ [Nm∕h] | 0,43 | 0,62 | 0,86 | 1,2 | 1,5 | 1,9 | 2,3 | 2,9 | 4,4 |
| | dry running: good ventilation oil lubrication: oil pray | | 0,49 | 0,71 | 0,99 | 1,38 | 1,73 | 2,19 | 2,65 | 3,34 | 5,06 |

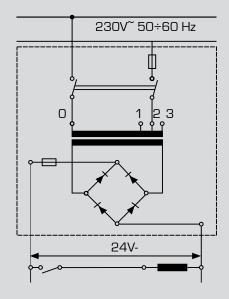
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Switching

Electromagnetic clutches are inductances. Engagement and disengagement are subject to the laws of induction, i. e. the induction current increases according to an e-function.





Shift speeds

Shift diagram: normal shifting

- t₁ and t₂ can be electrically influenced by taking appropriate measures
- it is advisable to use direct current for shifting
- when determining the size, the engage time is considered to be approximately 30% of the total acceleration time, which normally results in additional safety

Technical data

| Size | | | 16 | 21 | 22 | 24 | 26 | 28 | 31 | |
|--|----------------|--------|---------|---------|---------|---------|---------|---------|---------|---|
| shift speeds acc. to VDE 0580:2011-11 | t, | [msec] | 130/180 | 150/210 | 220/300 | 340/420 | 420/500 | 500/600 | 650/800 | normal excitation for oil and dry lubrication |
| | t ₂ | | 25/30 | 30/40 | 30/40 | 40/60 | 40/60 | 40/60 | 50/80 | |

i = induction current

t₁ = engage time

 M_{ij} = torque to be transferred / static torque

t₂ = disengage time

M_s = torque to be shifted

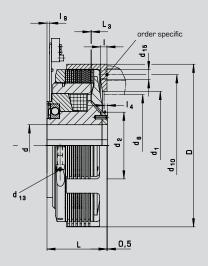
t₁₁ = response delay



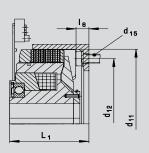
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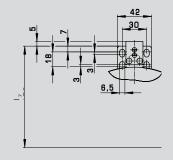
Clutch size



Typ 521._._.1 with normal drive ring



Typ 521._._2 with long drive ring



spool holder

Technical data

| Size | | | | 16 | 21 | 22 | 24 | 26 | 28 | 31 | 32 |
|---------------------------|-------------|----------------------------|------------------------|-------|-------|-------|-------|-------|-------|-------|-------|
| torque | oil running | M _s dyn | [Nm] | 60 | 100 | 200 | 400 | 600 | 800 | 1200 | 1600 |
| | | $M_{\dot{u}}$ stat | | 72 | 120 | 240 | 480 | 720 | 960 | 1450 | 1950 |
| | dry running | $\rm M_s$ dyn | | 80 | 135 | 270 | 540 | 800 | 1000 | 1600 | 2100 |
| | | $M_{\bar{u}}$ stat | | 105 | 175 | 350 | 700 | 1050 | 1300 | 2100 | 2700 |
| max. speed | oil running | | [min ⁻¹] | 3000 | 2500 | 2500 | 1500 | 1500 | 1500 | 1000 | 1000 |
| | dry running | | | 1500 | 1500 | 1500 | 1500 | 1000 | 1000 | 1000 | 1000 |
| input power | | | [W] | 33 | 43 | 61 | 85 | 99 | 111 | 112 | 144 |
| inertia | inside | | [10 ³ kgm²] | 3,18 | 4,7 | 13,4 | 24,5 | 50,3 | 86,3 | 140 | 328 |
| | outside | | | 1,13 | 3,55 | 7,83 | 15,3 | 25,3 | 47,3 | 75 | 150 |
| weight | | | [kg] | 2,8 | 5,3 | 7,5 | 12 | 16,5 | 22 | 31,5 | 48 |
| number of plates | inner plate | | | 6 | 7 | 7 | 6 | 6 | 6 | 6 | 6 |
| | outer plate | | | 6 | 7 | 7 | 6 | 6 | 6 | 6 | 6 |
| ballbearing DIN 625 | | | | 16007 | 16009 | 16012 | 16013 | 16016 | 16017 | 16020 | 16021 |
| bore | min. | Ød H7 | [mm] | 20 | 25 | 30 | 35 | 40 | 50 | 50 | 50 |
| keyway acc. to DIN 6885/1 | max. | | | 25 | 35 | 48 | 50 | 65 | 68 | 80 | 85 |
| dimensions | | ØD | [mm] | 115 | 140 | 166 | 195 | 214 | 240 | 264 | 295 |
| | | Ø d₁H7 | | 80 | 100 | 120 | 130 | 155 | 180 | 200 | 225 |
| | | Ø d ₂ | | 45 | 52 | 68 | 80 | 85 | 100 | 105 | 115 |
| | | Ø d ₈ | | 76 | 96 | 115 | 125 | 148 | 170 | 190 | 215 |
| | | | | 100 | 110 | 135 | 160 | 190 | 210 | 240 | 260 |
| | | Ø d ₁₀ | | 100 | 120 | 140 | 170 | 190 | 215 | 240 | 265 |
| | | Ø d ₁₁ | | 109 | 131 | 155 | 183 | 203 | 228 | 252 | 282 |
| | | Ø d ₁₂ | | 95 | 115 | 140 | 160 | 180 | 205 | 230 | 255 |
| | | Ø d ₁₃ DIN 6912 | | M5 | M5 | M6 | M6 | M6 | M6 | M6 | M8 |
| | | Ø d ₁₅ | | 4xM6 | 4xM8 | 4xM8 | 4xM12 | 4xM12 | 4xM12 | 4xM12 | 6xM16 |
| | | Ø d ₁₆ DIN 6912 | | M6 | M8 | M8 | M12 | M12 | M12 | M12 | M16 |
| ſſ | | L | | 53 | 63 | 67 | 73 | 81 | 90 | 101 | 110 |
| | | L ₁ | | 60,5 | 70 | 76,5 | 83 | 91 | 99 | 110 | 122 |
| ≠ Ø555 † R | | L ₃ | | 0,3 | 0,4 | 0,5 | 0,5 | 0,6 | 0,7 | 0,7 | 0,8 |
| | | | | 5 | 6 | 6,5 | 8 | 9 | 10 | 11 | 12 |
| | | l ₄ | | 2,5 | 3,5 | 3,5 | 4,5 | 4,5 | 5,5 | 5,5 | 6,5 |
| | | 6 max. | | 11 | 11 | 15 | 16 | 16 | 16 | 18 | 21 |
| <u> </u> | | I ₇ | | 82 | 97,5 | 108 | 123 | 132,5 | 144 | 158,5 | 168 |
| | | l _g | | 4 | 4 | 4 | 4 | 4 | 4 | 5 | 5 |

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Operating mode

Mönninghoff electromagnet multiple-disc clutches are avaible in two operating modes

- Type 521._.1._ for dry running
- Type 521._.2._ for oil running

To reduce the engage time, fast excitation can be achieved by applying up to three times the rated voltage. When oil is used and particularly if the oil is cooled internally, the rise time can be affected considerably and may double or triple (observe oil instructions).

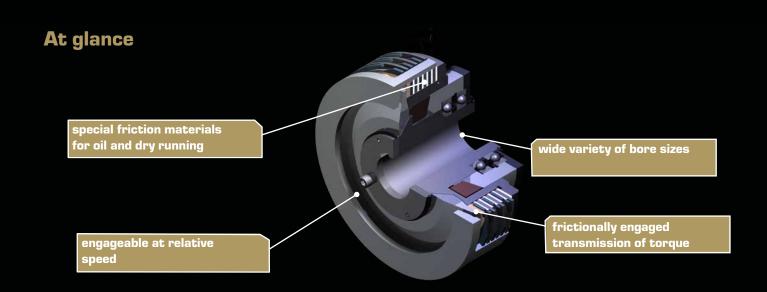
Use oil with a viscosity up to 25 x 10^{-6} m²·s⁻¹ by 50 °C (3°E / 50 °C).

Voltage

- standard voltage is 24 Vdc dirrect current
- special voltages as a example 48 Vdc on request

Technical characteristics

- the positioning of the discs outside the magnetic field permits the use of special friction materials for both oil and dry lubrication
- the expanding springs of the outside discs open the disc stack when the coil is switched off.
 Consequently, friction and wear in neutral are negligible
- the adjustment of the air gap is easily accessible



System solutions

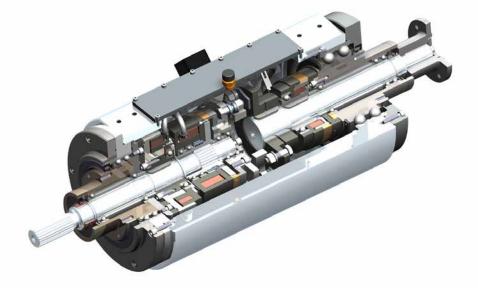


You need more?

Mönninghoff couplings can be combined with a variety of many other power transmission elements. Such complex high-tech systems can solve any application-specific tasks and can fulfill any customer-specific wishes.



In many cases, a combination of different drive elements is needed to solve the applications particular problems and difficulties. Being not just supplier but technological partner to our customers, our extensive engineering is part of extraordinary and challenging power transmission projects.







 $\, \odot \,$ 08/2022 | Electromagnetic multiple-disc clutch - Type 521 | EN Subject to technical alterations.

